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# Fifth Semester B.E. Degree (CBCS) Examination Turbomachines 

Time: 3 hrs .
Max. Marks: 80

## Note: Answer any FIVE full questions, choosing one full question from each module.

## MODULE - I

1 a Define turbomachine. Give a comparison between turbomachines and positive (08 Marks) displacement machines
b A single stage centrifugal pump works against a height of 30 m , running at ( 08 Marks) 2000 rpm , supplies $3 \mathrm{~m}^{3} / \mathrm{s}$ and has an impeller diameter of 300 mm . Calculate (a) the number of stages and (b) the diameter of each impeller required to pump $6 \mathrm{~m}^{3} / \mathrm{s}$ of water to a height of 220 m when running at 1500 rpm .

## OR

Define total to total, total to static, static to static and static to total efficiencies for power developing and power consuming turbomachines and write the T-s Diagrams.
b Total to total efficiency for a power absorbing turbomachine handling liquid water (08 Marks) of standard density is $70 \%$. Suppose the total pressure of water increased by 4 bar, evaluate (a) the isentropic change in total enthalpy (b) the actual change in total enthalpy (c) the change in total temperature of the water and (d) the power input to the water, flow rate is $30 \mathrm{~kg} / \mathrm{s}$.

## MODULE - II

3 a In a certain turbomachine, the blade speed at exit is twice that at inlet $\left(u_{2}=2 u_{1}\right)$, the meridian component of fluid velocity at inlet is equal to that at exit and the blade angle at inlet is $45^{\circ}$. Show that the energy transfer per unit mass and degree of reaction are given by $\frac{E}{m}=-2 V_{m 1}^{2}\left(2-\cot \beta_{2}\right) \quad$ and $R=\frac{\left(\cot \beta_{2}+2\right)}{4}$
b At a stage of $50 \%$ reaction axial flow turbine running at 3000 rpm , the mean blade ( 08 Marks) diameter is 68.5 cm . If the maximum utilization factor for the stage is 0.915 , Calculate (a) the inlet and outlet absolute velocities and (b) the power output. Also, find the power developed for a steam flow rate of $15 \mathrm{~kg} / \mathrm{s}$.

## OR

4 a Derive the theoretical head capacity relation in case of centrifugal (08 Marks) pump $\quad H=\frac{U_{2}^{2}}{g_{c}}-\frac{U_{2}^{2} Q \cot \beta_{2}}{A_{2} g_{c}}$. Discuss the effect of blade angle at outlet on head.
b Draw the inlet and outlet triangles for an axial flow compressor for which given (1)
(08Marks) Degree of reaction $=0.5$ (2) inlet blade angle $=40^{0}$ axial velocity of flow which is constant throughout $=125 \mathrm{~m} / \mathrm{s}(4)$ RPM $=6500$ (5) Radius $=0.2 \mathrm{~m}$. Calculate the power required in kW at an air flow rate $=15 \mathrm{~kg} / \mathrm{s}$. Find fluid angles at inlet and outlet. Blade speed is same at exit and inlet.

## MODULE - III

5 a Derive the condition for maximum efficiency of an impulse turbine and show that (08 Marks) the maximum efficiency is $\cos ^{2} \alpha$.
b Steam issues from nozzle to a de Laval turbine at a velocity of $1000 \mathrm{~m} / \mathrm{s}$. The ( 08 Marks) nozzle angle is $20^{\circ}$. The mean blade velocity is $400 \mathrm{~m} / \mathrm{s}$. the blades are symmetrical. The mass flow rate is $1000 \mathrm{~kg} / \mathrm{h}$, friction factor is 0.8 , and nozzle efficiency is 0.95 . Calculate Blade angle, Axial thrust, and Power developed, Blade efficiency, Stage efficiency.
a Show that the maximum diagram efficiency of a stage of a reaction turbine is given by the expression $\emptyset=\frac{2 \cos ^{2} \alpha_{1}}{1+\cos ^{2} \alpha_{1}}$
b In a Curtis steam turbine stage there are two rows of moving blades with equiangular rotors. Steam enters the first rotor at an angle of $20^{\circ}$ each and the second rotor at an angle of $32^{\circ}$ each. The absolute velocity of steam as it enters the first rotor is $530 \mathrm{~m} / \mathrm{s}$ and the blade velocity coefficient is 0.9 in the first rotor, 0.91 in stator, and 0.93 in the second rotor. If the final discharge should be axial,
Compute (a) the power output for a steam flow rate of $3.2 \mathrm{~kg} / \mathrm{s}$ and the axial thrust.

## MODULE - IV

a Derive an expression for maximum efficiency of a pelton wheel.
b A Pelton wheel has a water supply rate of $5 \mathrm{~m}^{3} / \mathrm{s}$ at a head of 256 m and runs at 500rpm. Assuming a turbine efficiency of 0.85 , a coefficient of velocity for nozzle as 0.985 , speed ratio of 0.46 , calculate (a) the power output, (b)the specific speed. OR
a With a neat sketch explain the working principle of Kaplan turbine.
b An inward flow reaction turbine with a supply of $0.6 \mathrm{~m}^{3} / \mathrm{s}$ under a head of 15 m develops 75 kw at 400 rpm . The inner and outer diameter of the runner are 40 cm and 65 cm respectively. Water leaves the exit of the turbine at $3 \mathrm{~m} / \mathrm{s}$ calculate the hydraulic efficiency and the inlet blade angles. Assume radial discharge and width to be constant.

## MODULE -V

a Explain the phenomenon of cavitation in a centrifugal pump? What are the effects? How do you prevent cavitation?
b A centrifugal pump delivers 501/s of water per second against a total head of 24 m at 1500 rpm . The velocity of flow is maintained constant at $2.4 \mathrm{~m} / \mathrm{s}$ and blades are curved backward at $30^{\circ}$ to tangent at exit. The inner diameter is half of the outer diameter, if the Manometric efficiency is $80 \%$. Find the blade angle, and power required to pump.

## OR

a Draw a sketch of an axial flow compressor with inlet guide vane and explain the working principle of the compressor
b An air compressor has eight stages of equal pressure ratio 1.35. The flow rate through the compressor and its overall efficiency are $50 \mathrm{~kg} / \mathrm{s}$ and $82 \%$ respectively. If the conditions of air at entry are 1.0bar and 400c Determine a) the state of air at the compressor exit b) polytropic efficiency

